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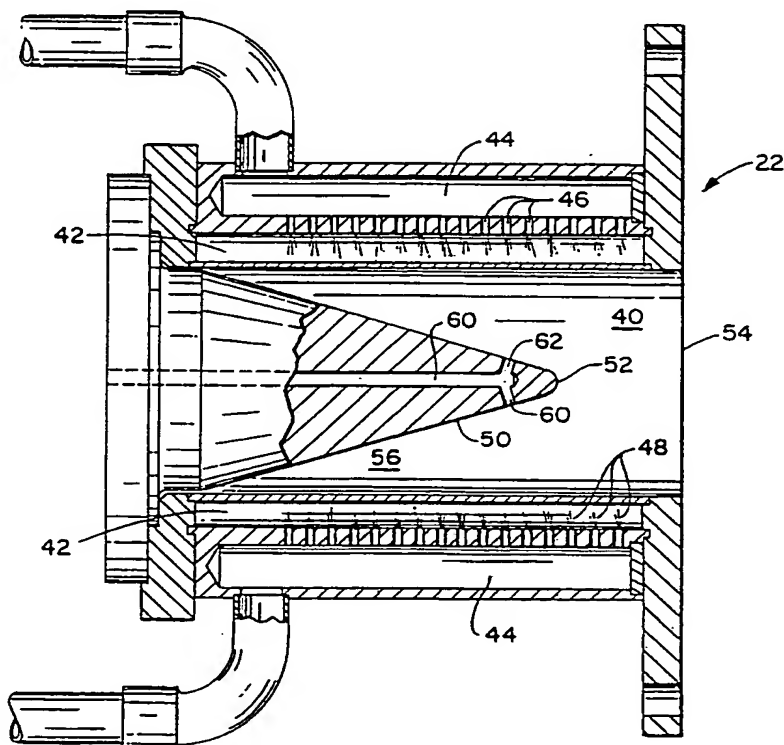
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<p>(21) International Application Number: PCT/US92/10269 (22) International Filing Date: 20 November 1992 (20.11.92) (30) Priority data: 841,942 26 February 1992 (26.02.92) US (71) Applicant: UNITED TECHNOLOGIES CORPORATION [US/US]; United Technologies Building, Hartford, CT 06101 (US). (72) Inventor: HU, Aaron, S. ; 39 Allendale Road, Hartford, CT 06106 (US). (74) Agent: KOCHEY, Edward, L., Jr.; United Technologies Corporation, Patent Department, United Technologies Building, Hartford, CT 06101 (US).</p>		<p>(81) Designated States: JP, European patent (AT, BE, CH, DE, DK, ES, FR, GB, GR, IE, IT, LU, MC, NL, SE).  Published With international search report.</p>

(54) Title: PREMIX GAS NOZZLE

(57) Abstract

A premix gas nozzle has longitudinal tangential entrance slots (42) to a cylindrical chamber (40). There is an axially increasing flow area (56) toward the chamber outlet (54), with pilot fuel (60, 62) centrally introduced near the outlet. A lean mix low NO<sub>x</sub> fuel nozzle is thereby stabilized.



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## Description

## Premix Gas Nozzle

## Technical Field

The invention relates to fuel nozzles for low NOx  
5 combustion and in particular to the stabilization  
thereof.

## Background of the Invention

Combustion at high temperature leads to the  
formation of NOx, or oxides of nitrogen, because of the  
10 combination of oxygen with nitrogen at high  
temperature. This is a notorious pollutant and much  
effort is being put forth to reduce the formation of  
NOx.

One solution has been to premix the fuel with  
15 excess air whereby all of the combustion occurs with a  
local high excess air and therefore at a relatively low  
temperature. Such combustion, however, can lead to  
instability and incomplete combustion.

This problem is exacerbated in gas turbine  
20 engines. Once the proper lean mix is set for proper  
full load operation, low load operation must be  
considered. At decreasing loads the airflow decreases  
less than the fuel flow, leading to even leaner  
mixtures. The air temperature also decreases.  
25 Accordingly, flame stability and combustion efficiency  
(percentage of fuel burnt) becomes an increasing  
problem.

## Summary of the Invention

Gas and air are mixed at a tangential entrance  
30 through longitudinal slots in a cylindrical chamber. A

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center cone provides an increasing axial flow area toward the chamber outlet.

The gas swirl within the chamber completes the air and gas mixing. Additional gas is supplied as pilot fuel on the central axis of the chamber near the outlet.

This pilot fuel remains in the core. As it leaves the chamber it is met with high temperature recirculating products from the flame. These products are primarily hot air because of the high localized air/fuel ratio. Local self ignition maintains the flame stability. It has also been found to increase the combustion efficiency.

As load is decreased pilot fuel is maintained constant, or at least reduced less than the main fuel. This increase in local combustion is acceptable without increasing NOx since the air temperature itself is decreasing at these low loads.

#### 20 Brief Description of the Drawings

Figure 1 is a schematic of a gas turbine engine and combustor;

Figure 2 is a sectional side view of the burner;

Figure 3 is a sectional axial view of the burner;

25 Figure 4 is a sectional axial view taken at 90° from Figure 3; and

Figure 5 is a sectional axial view of an alternate burner embodiment.

#### 30 Description of the Preferred Embodiment

The Figure 1 schematic illustrates a gas turbine engine with compressor 10 supplying compressed air to combustor 12. Gas which is fueled through gas supply line 14 provides fuel for combustion within the combustor with the gaseous products passing through turbine 16.

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Referring to Figure 2, combustor 12 is surrounded by combustor liner 18 and has in the upstream face 20 a plurality of circumferentially spaced burners 22. The structure is sized such that of the incoming airflow 24 from the compressor 35 percent of this flow passes as dilution air 26 around a burner with the majority of this passing as cooling air 28 through the combustion liner. 65 percent of this airflow passes as combustion supporting air 30 through the burner.

From the fuel header 14 the main gas flow is supplied through line 32 and controlled by valve 34. A pilot flow of gas passes through pilot line 36 being controllable by valve 38.

Referring to Figures 3 and 4, burner 22 is comprised of a substantially cylindrical axially extending chamber 40. Two longitudinally extending slots 42 are located with the walls tangent to the inner wall of the cylindrical chamber. Combustion supporting airflow 30 passes through these slots establishing a whirling action in chamber 40. The main gas flow line 32 is divided to supply two gas distribution manifolds 44 located adjacent the air inlet slot 42. A plurality of holes 46 are located along the length of manifold 44. These distributively inject gas as a plurality of streams 48 into the airflow passing into the slot. The gas and air continue mixing as the mixture swirls through chamber 40.

Centrally located within the chamber 40 is a cone 50 with its base toward the upstream end of the chamber and its apex 52 toward the outlet 54 end of the chamber. Resulting flow area 56 therefore increases toward the outlet of the chamber so that the mixture of air and gas passing axially along the chamber maintains a somewhat constant velocity. This deters flashback from the flame into the upstream end of the chamber.

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The substantially cylindrically chamber 40 is formed by two semi-cylindrical walls 58 each having its axis offset from one another to form the slots 42.

A gas pilot tube 60 passes through the center of  
5 the cone with pilot discharge openings 62 at or adjacent the apex 52 of the cone. This location should be within 25 percent of the length of the chamber 40 from the outlet 54 of the chamber. The objective is to introduce the additional gas flow centrally of the  
10 swirling air/gas mixture, but not to mix it in with the air/gas mixture. This is aided by the fact that the incoming gas is lighter than the air or air/gas mixture.

In full load operation of the gas turbine engine,  
15 between 4 and 6 percent of the total gas flow may be supplied through the pilot openings 62 without increasing the NOx. In most cases the pilot is not needed for stability at the high load. The flow, however, cools the nozzle, and avoids operational  
20 complexity of turning the pilot on when load is reduced. Pilot operation is therefore preferred, though not required at full load.

As load is reduced on the gas turbine engine, the overall airflow drops less rapidly than the gas flow.  
25 Since the relationship of the airflow between the combustion air and the dilution air is set by the physical design of the structure, it remains constant. The mixture in the combustion zone therefore becomes increasingly lean. The preferred operation is to  
30 decrease load by closing down on valve 34 while leaving valve 38 open. This increases the proportion of fuel introduced through the pilot. At this same time, however, the air temperature from the compressor decreases. The additional temperature because of the  
35 higher concentration of pilot fuel is acceptable

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without increasing NOx because of this overall temperature decrease.

It is understood that during test operation it may be found that some other manipulation of valve 38 is preferred rather than to maintain it in a fixed position. It nonetheless should produce an increasing percentage of the fuel through the pilot during load decrease.

Figure 5 illustrates a section through an alternate nozzle embodiment showing chamber 40 and cone 50. Three inlet slots 72 are provided for the air inlet while the main gas flow passes through gas manifolds 74 and ejecting through holes 76 into slot 72.

Flame stability is achieved without NOx increase at reduced loads.

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## Claims

1. A low NOx burner for a gas turbine engine, comprising:

a substantially cylindrical burner chamber having an axially extending chamber wall, and having an upstream end and an outlet end;

at least one longitudinally extending slot in the wall of said cylindrical chamber having a slot wall tangential to said chamber wall;

supply means for supplying air through said slot;

a gas distribution manifold located adjacent said slot and having a plurality of axially space openings for delivering gas into the airflow as it passes into said slot;

a conical body located in said chamber on the axis of said chamber with the base of said conical body at the upstream end of said chamber and the apex of said conical body toward the outlet end of said chamber; and

a gas pilot tube having a discharge opening through said conical body at the apex end.

2. A burner as in claim 1 comprising also:

said substantially cylindrical chamber formed of a plurality of partial cylinders having the axis of each cylinder offset from the axis of the others, whereby said slot is formed between the walls of adjoining partial cylinders.

3. A burner as in claim 2 comprising also:

the number of partial cylinders being two.

4. A burner as in claim 1 comprising also:

said gas pilot tube having a plurality of circumferentially spaced discharge openings around the



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periphery of said conical body at or slightly upstream  
5 of the apex of said conical body.

5. A burner as in claim 1 comprising also:  
said discharge opening through said conical body  
being located within 25 percent of the axial length of  
said chamber from the outlet of said chamber.

6. A burner as in claim 3 comprising also:  
said gas pilot tube having a plurality of  
circumferentially spaced discharge openings around the  
periphery of said conical body at or slightly upstream  
5 of the apex of said conical body.

7. A burner as in claim 4 comprising also:  
said discharge opening through said conical body  
being located within 25 percent of the axial length of  
said chamber from the outlet of said chamber.

8. A burner as in claim 6 comprising also:  
said discharge opening through said conical body  
being located within 25 percent of the axial length of  
said chamber from the outlet of said chamber.

9. A method of burning gas in the combustor of a gas  
turbine engine with a premixing type of combustion,  
comprising:

tangentially introducing combustion air into a  
5 substantially cylindrical chamber having increased  
axial flow area toward an outlet end of said  
substantially cylindrical chamber;  
distributively injecting a main gas flow into said  
combustion air at the entrance to said substantially  
10 cylindrical chamber;

burning said main gas flow at the outlet of said  
substantially cylindrical chamber; and

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introducing a pilot gas flow into said gas chamber at the central axes of said chamber.

10. The method of claim 9 including also:

introducing said pilot gas flow at a location within 25 percent of the axial length of said chamber from the outlet of said chamber.

11. The method of claim 9 comprising also:

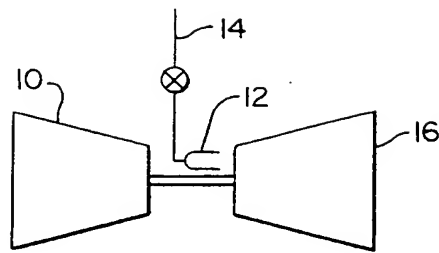
at maximum output of said gas turbine engine introducing as pilot gas flow between 4 and 6 percent of the total of said pilot gas flow and said main gas flow; and

increasing the percentage of said pilot gas flow as a percentage of the total flow at outputs below said maximum amount.

12. The method of claim 10 comprising also:

at maximum output of said gas turbine engine introducing as pilot gas flow between 4 and 6 percent of the total of said pilot gas flow and said main gas flow; and

increasing the percentage of said pilot gas flow as a percentage of the total flow at outputs below said maximum amount.



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FIG. 1

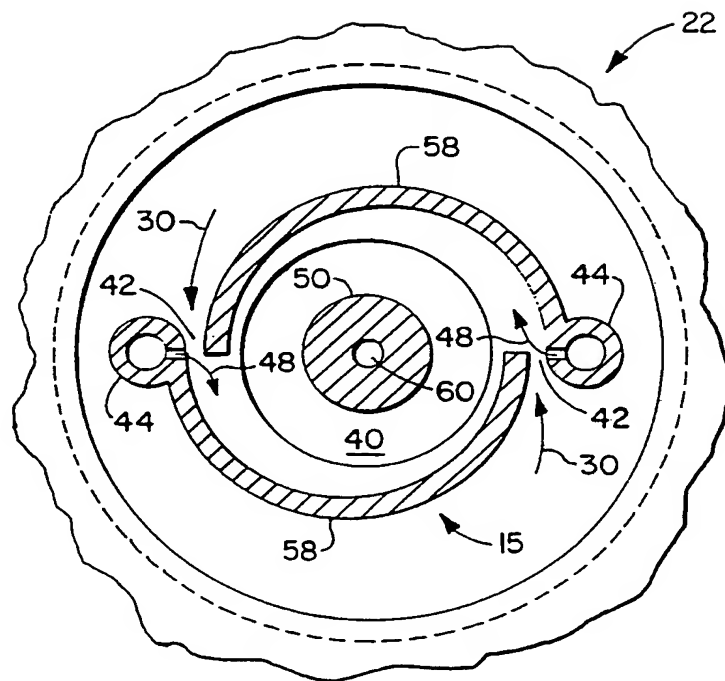


FIG. 3

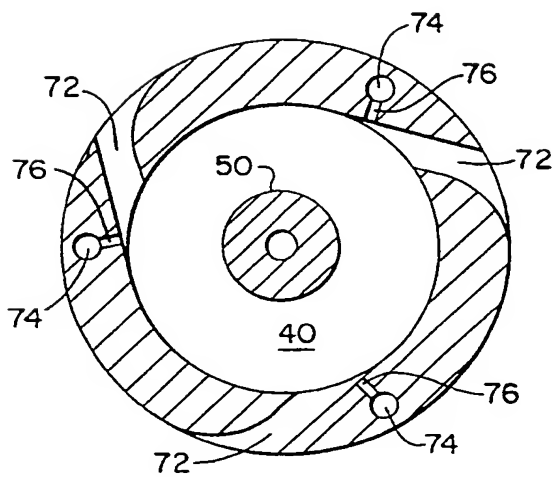


FIG. 5

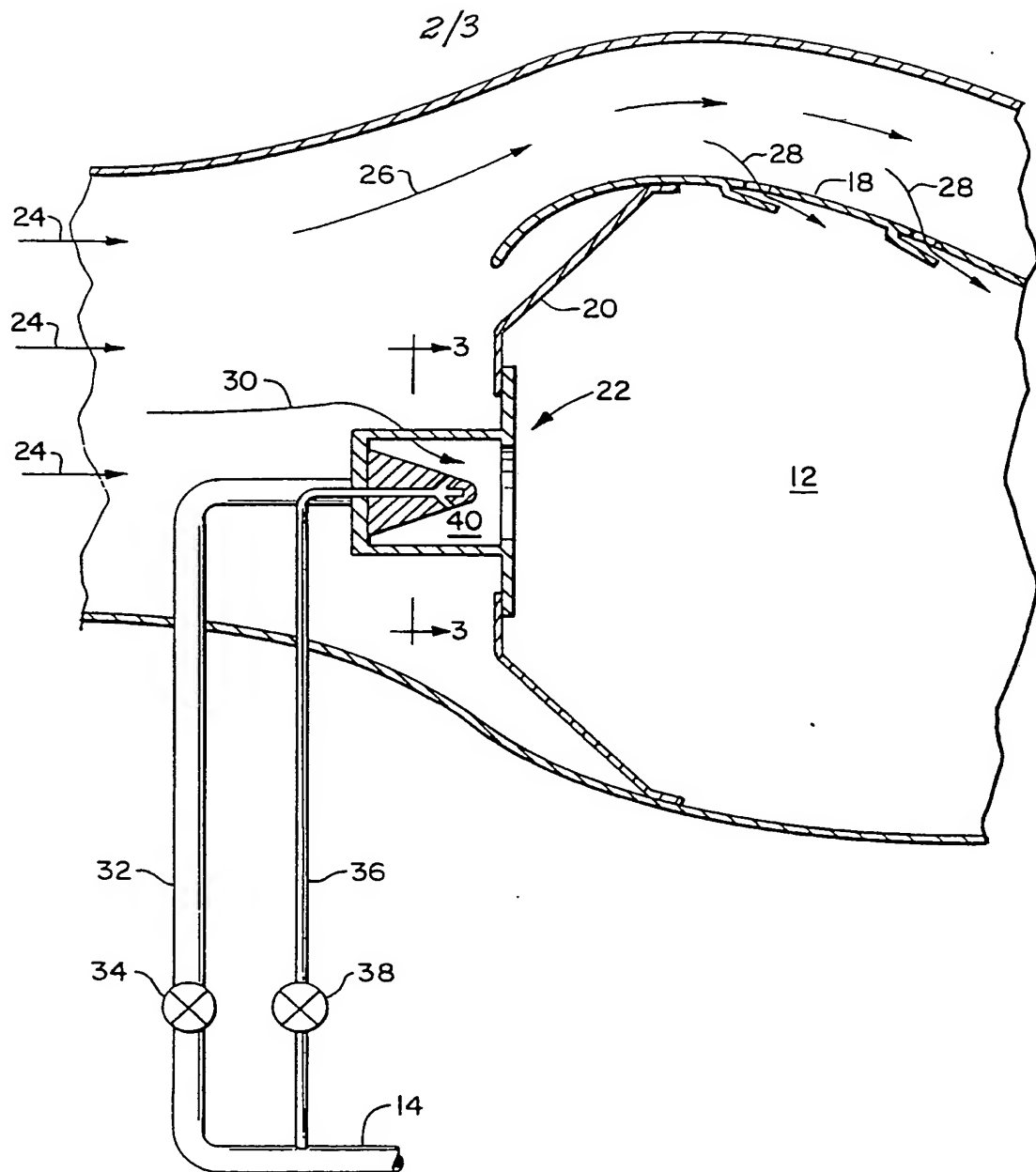
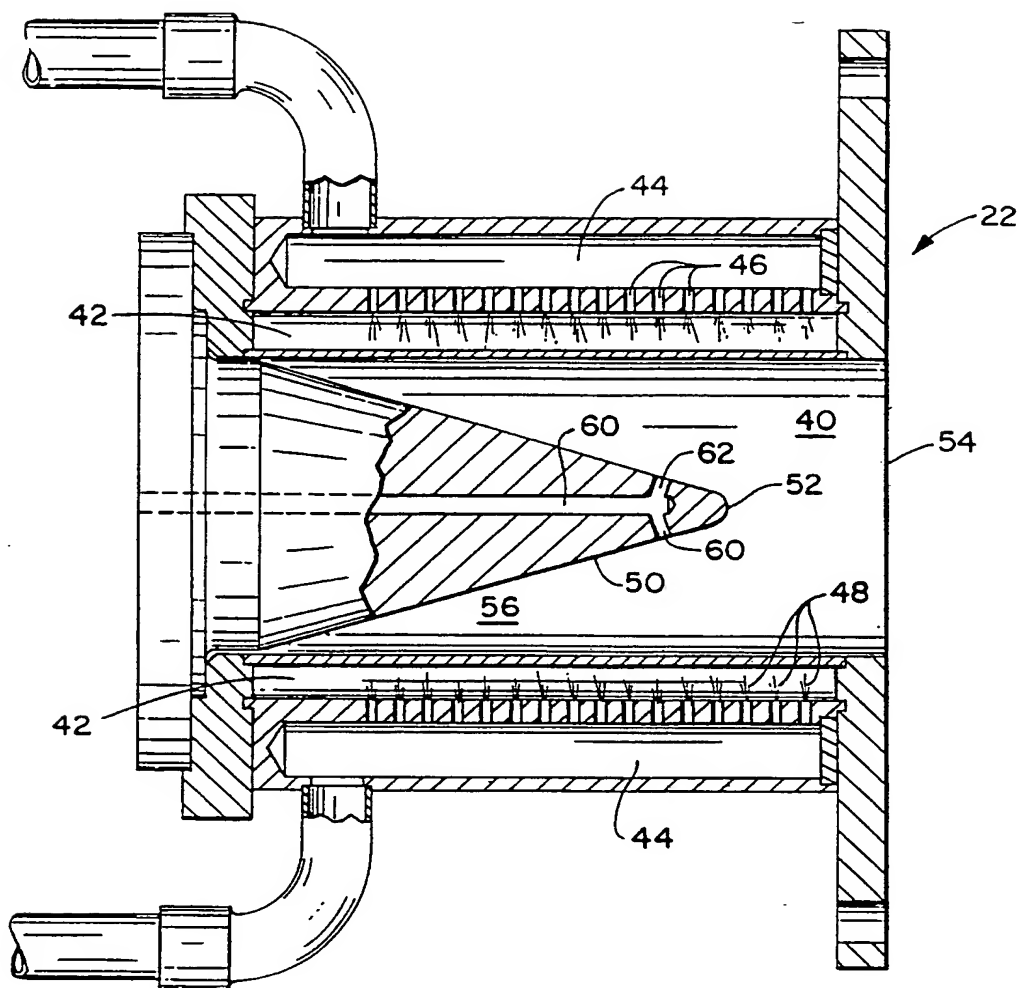


FIG. 2

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FIG. 4

## INTERNATIONAL SEARCH REPORT

International Application No

PCT/US 92/10269

<b>I. CLASSIFICATION OF SUBJECT MATTER</b> (if several classification symbols apply, indicate all) <sup>6</sup>		
According to International Patent Classification (IPC) or to both National Classification and IPC		
Int.Cl. 5 F23D14/02; F23D14/62; F23C7/00		
<b>II. FIELDS SEARCHED</b>		
Minimum Documentation Searched <sup>7</sup>		
Classification System	Classification Symbols	
Int.Cl. 5	F23C ; F23D ; F23R	
Documentation Searched other than Minimum Documentation to the Extent that such Documents are Included in the Fields Searched <sup>8</sup>		
<b>III. DOCUMENTS CONSIDERED TO BE RELEVANT<sup>9</sup></b>		
Category <sup>10</sup>	Citation of Document, <sup>11</sup> with indication, where appropriate, of the relevant passages <sup>12</sup>	Relevant to Claim No. <sup>13</sup>
A	EP,A,0 433 790 (ASEA BROWN BOVERI) 26 June 1991 see abstract; figures ---	1,9
A	EP,A,0 310 327 (EXXON RESEARCH AND ENG.) 5 April 1989 see column 5, line 40 - column 6, line 7 see abstract; figures 1,2 -----	1,9
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**ANNEX TO THE INTERNATIONAL SEARCH REPORT  
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